

## 第一章 绪论

### 习题 1-1

解:

$$500L = 0.5m^3 \rho Q (\beta_2 v_c - \beta_1 v_1) = P_1 - P_c - R$$

$$\rho = \frac{m}{V} = \frac{6795}{0.5} = 1.359 \times 10^4 \text{ kg/m}^3$$

$$\gamma = \rho g = 1.359 \times 10^4 \times 9.8 = 1.33 \times 10^5 \text{ N/m}^3$$

### 习题 1-2

解:

$$\rho = \frac{\gamma}{g} = 9.8 = 998.9 \text{ kg/m}^3$$

$$\nu = \frac{\mu}{\rho} = \frac{1.002 \times 10^{-3}}{998.9} = 1.0031 \times 10^{-6} \text{ m}^2/\text{s}$$

### 习题 1-3

解:

体积弹性系数

$$K = \frac{1}{\beta} = -\frac{dp}{dV/V}$$

根据已知条件代入数据得到

$$2.18 \times 10^9 = -\frac{dp}{-1\%}$$

求得

$$dp = 2.18 \times 10^7 \text{ N/m}^2$$

## 第二章 水静力学

### 习题 2-2

解：由图可以看出过石油和甘油交界面的水平面为甘油液面的等压面，根据等压面性质可得，

$$p_{\text{左}} = p_{\text{表}} + \gamma_{\text{石油}} \times (7-3)$$

$$p_{\text{右}} = p_a + \gamma_{\text{甘油}} \times (9-3)$$

$$p_{\text{左}} = p_{\text{右}} \Rightarrow p_{\text{表}} + \gamma_{\text{石油}} \times (7-3) = p_a + \gamma_{\text{甘油}} \times (9-3)$$

$$\text{则, } p_{\text{表}} = p_a + \gamma_{\text{甘油}} \times (9-3) - \gamma_{\text{石油}} \times (7-3)$$

$$\text{代入数据得, } p_{\text{表}} = 101.325 + 12.25 \times (9-3) - 8.17 \times (7-3) = 142.145 \text{ kPa}$$

### 习题 2-5

解：

通过 U 形水银测压计左侧液面做水平面，该水平面既是水面的等压面，又是水银液面的等压面，假设液面绝对压强为  $p'_{\text{液面}}$ ，相对压强为  $p_{\text{液面}}$  根据等压面性质可知：

$$p'_{\text{液面}} = p_a + \gamma_{\text{水银}}(h_1 - h_2) - \gamma_{\text{水}}(H - h_2)$$

代入数据

$$p'_{\text{液面}} = 101.325 + 133.23 \times (2.8 - 2.4) - 9.8 \times (5.5 - 2.4) = 124.237 \text{ kPa}$$

液面相对压强为

$$p_{\text{液面}} = p'_{\text{液面}} - p_a = 124.237 - 101.325 = 22.912 \text{ kPa}$$

## 习题 2-7

解:

由图可知, 过容器内水面的水平面为等压面, 根据等压面性质可得到:

$$p'_0 + \gamma_{\text{水}} h_0 = p'_c$$

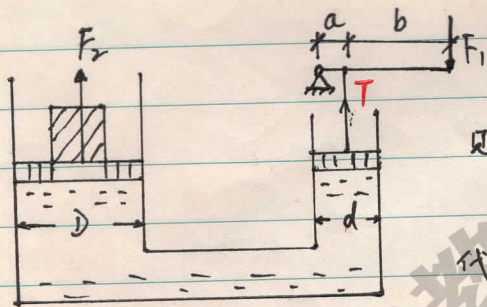
$$\text{则, } p'_c = p'_0 + \gamma_{\text{水}} h_0 = 86.5 + 9.8 \times 2 = 106.1 \text{ kPa}$$

根据等压面性质还可得到:

$$p'_c = p_a + \gamma_{\text{水}} h$$

$$\text{则, } h = \frac{p'_c - p_a}{\gamma_{\text{水}}} = \frac{106.1 - 101.325}{9.8} = 0.487 \text{ m}$$

2-9.



解: 设小活塞作用于杠杆上的力为  $T$ .

则对支点列合力矩为

$$F_1 \times (a+b) = T \times a$$

代入数据:

$$186 \times (20+80) = T \times 20$$

$$\Rightarrow T = 930 \text{ N}$$

$$S_{\text{小活塞}} = \pi \left(\frac{d}{2}\right)^2 = 3.14 \times \left(\frac{0.06}{2}\right)^2 = 0.002826 \text{ m}^2$$

$$p_{\text{小活塞}} = \frac{T}{S_{\text{小活塞}}} = \frac{930}{0.002826} = 329.09 \text{ kPa}$$

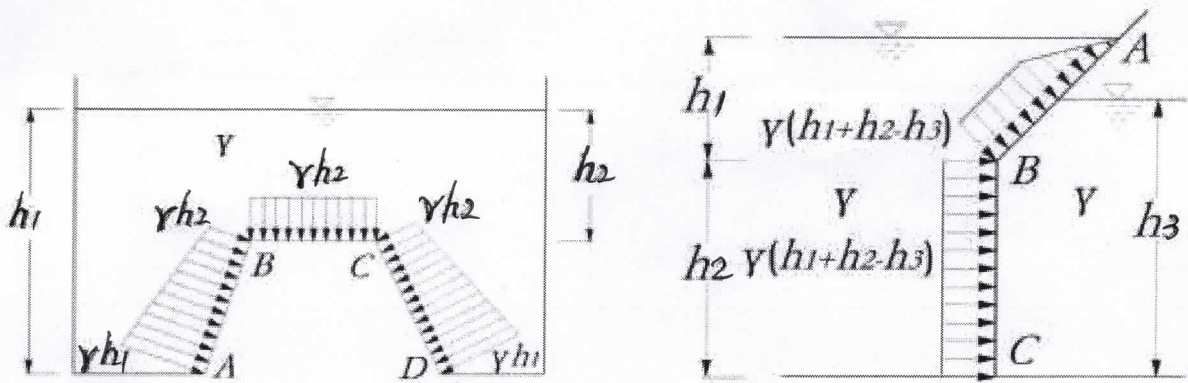
根据等压面性质有  $p_{\text{大活塞}} = p_{\text{小活塞}} = 329.09 \text{ kPa}$

$$\text{即, } p_{\text{大活塞}} = \frac{F_2}{S_{\text{大活塞}}} = \frac{836}{S_{\text{大活塞}}} = 329.09 \text{ kPa} \Rightarrow S_{\text{大活塞}} = 0.0254 \text{ m}^2$$

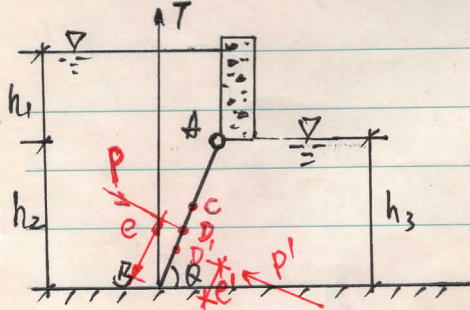
$$\text{即, } S_{\text{大活塞}} = \pi \left(\frac{D}{2}\right)^2 = 3.14 \times \left(\frac{D}{2}\right)^2 = 0.0254$$

$$\Rightarrow D = 17.98 \text{ cm}$$

习题 2-14(b、d)



2-16



解: (1) 当下游无水时,  
由于是矩形闸门, 所以由式(2-26)可得  
作用在闸门上的力为:

$$\begin{aligned}
 P &= \gamma h_c A = 9.8 \times \left( h_1 + \frac{h_2}{2} \right) \times A \\
 &= 9.8 \times \left( h_1 + \frac{h_2}{2} \right) \times b \times \frac{h_2}{\sin 60^\circ} \\
 &= 9.8 \times \left( 1.2 + \frac{1.85}{2} \right) \times 3 \times \frac{1.85}{\frac{\sqrt{3}}{2}} \\
 &= 133.46 \text{ kN}
 \end{aligned}$$

则作用点距B点的距离为

$$e = \frac{L}{3} \cdot \frac{2h_1' + h_2'}{h_1' + h_2'} = \frac{\frac{h_2}{\sin 60^\circ}}{3} \cdot \frac{2h_1 + (h_1 + h_2)}{h_1 + (h_1 + h_2)} = \frac{\frac{1.85}{\sin 60^\circ}}{3} \cdot \frac{2 \times 1.2 + (1.2 + 1.85)}{1.2 + (1.2 + 1.85)} = 0.913 \text{ m}$$

$$h_1' = h_1$$

$$h_2' = h_1 + h_2$$

则作用点距A点的距离为  $DA = AB - e = \frac{h_2}{\sin 60^\circ} - e = \frac{1.85}{\sin 60^\circ} - 0.913 = 1.223 \text{ m}$

对A点列力矩方程有:

$$T \times AB \cos 60^\circ = P \times DA + G \times \frac{1}{2} AB \cos 60^\circ$$

代入数据:

$$T \times \frac{h_2}{\sin 60^\circ} \times \cos 60^\circ = 133.46 \times 1.223 + 9.8 \times \frac{1}{2} \times \frac{h_2}{\sin 60^\circ} \times \cos 60^\circ$$

$$\text{即 } T \times \frac{1.85}{\sin 60^\circ} \times \cos 60^\circ = 133.46 \times 1.223 + 9.8 \times \frac{1}{2} \times \frac{1.85}{\sin 60^\circ} \times \cos 60^\circ \Rightarrow T = 157.7 \text{ kN}$$

(2). 当下游有水时.

设下游对闸门的压力为  $P'$ . 作用点为  $D'$ .

则有:

$$\begin{aligned} P' &= \gamma h_c A = 9.8 \times \frac{h_3}{2} \times b \times \frac{h_2}{\sin 60^\circ} \\ &= 9.8 \times \frac{1.85}{2} \times 3 \times \frac{1.85}{\sin 60^\circ} \\ &= 58.1 \text{ kN} \end{aligned}$$

$$e' = \frac{1}{3} = \frac{\frac{h_3}{\sin 60^\circ}}{3} = \frac{\frac{1.85}{\sin 60^\circ}}{3} = 0.712 \text{ m.}$$
 因为右边压强分布图是三角形.

$$\text{则 } D'A = AB - e' = \frac{h_2}{\sin 60^\circ} - e' = \frac{1.85}{\sin 60^\circ} - 0.712 = 1.424 \text{ m}$$

对A点列力矩方程有:

$$T \times AB \cos 60^\circ + P' \times D'A = P \times DA + G \times \frac{1}{2} AB \cos 60^\circ$$

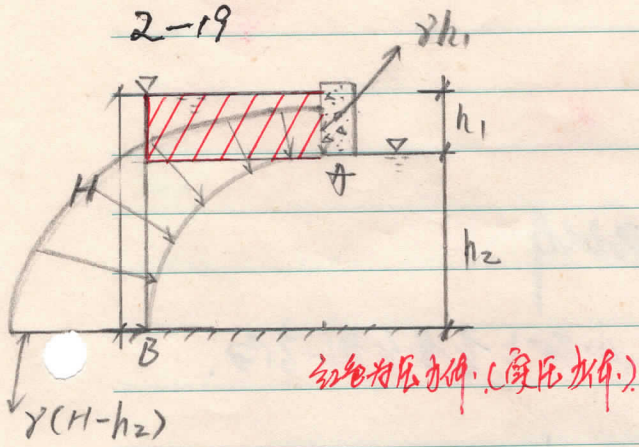
$$\text{即 } T \times \frac{h_2}{\sin 60^\circ} \times \cos 60^\circ + 58.1 \times 1.424 = 133.46 \times 1.223 + 9.8 \times \frac{1}{2} \times \frac{h_2}{\sin 60^\circ} \times \cos 60^\circ$$

代入数据:

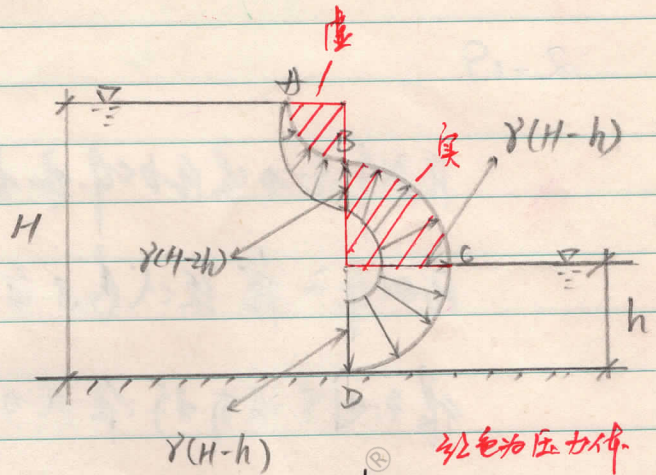
$$T \times \frac{1.85}{\sin 60^\circ} \times \cos 60^\circ + 58.1 \times 1.424 = 133.46 \times 1.223 + 9.8 \times \frac{1}{2} \times \frac{1.85}{\sin 60^\circ} \times \cos 60^\circ$$

$$\Rightarrow T = 80.27 \text{ kN}$$

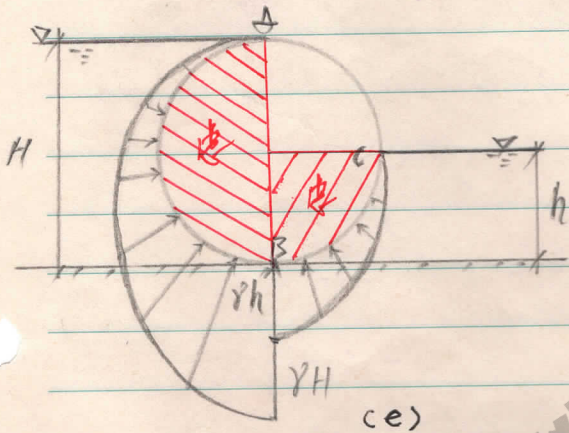
2-19



(c)

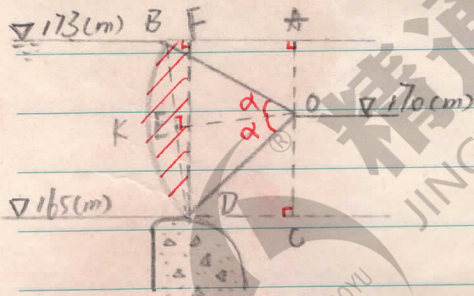


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2-20



解：闸前水深为

$$h = 173 - 165 = 8 \text{ m}$$

则水平总力  $P_y$  为：

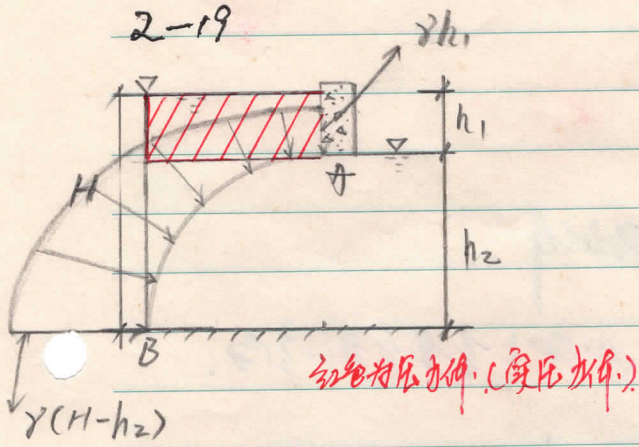
$$\begin{aligned} P_y &= \gamma h c A_y = 9.8 \times \frac{h}{2} \times h \times b \\ &= 9.8 \times \frac{8}{2} \times 8 \times 10 \\ &= 3136 \text{ kN} \end{aligned}$$

铅垂力为压力体 FBD 内的水体。受压水体的体积等于面积  $S_{BDK}$  与三角形 BFD 的面积  $S_{\triangle BFD}$  之和再乘以闸孔  $b$ 。

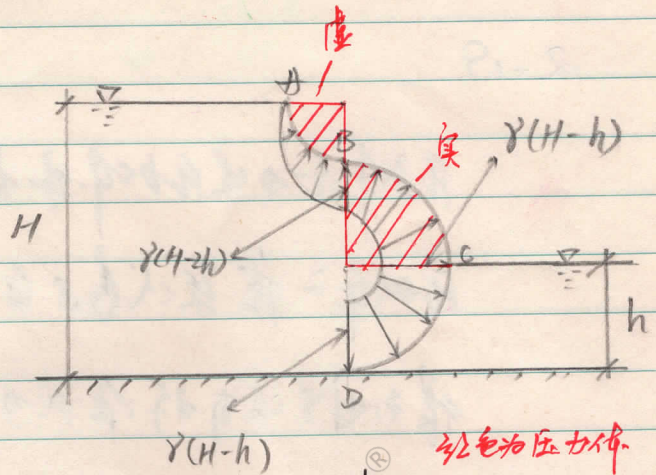
如图中所示：  $AO = 3 \text{ m}$      $OC = 5 \text{ m}$      $OB = OD = R = 8 \text{ m}$      $FD = 8 \text{ m}$

$$\text{则 } AB = \sqrt{OB^2 - AO^2} = \sqrt{8^2 - 3^2} = 7.42 \text{ m}$$

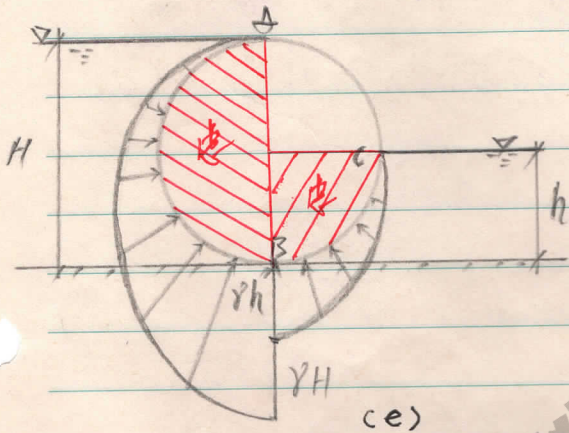
2-19



(c)

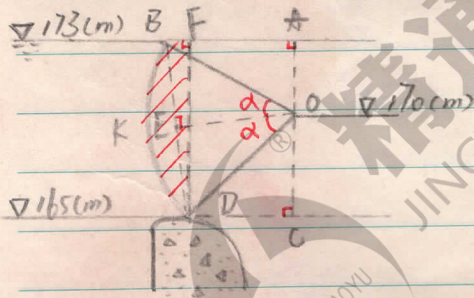


(cd)



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2-20



解：闸前水深为

$$h = 173 - 165 = 8 \text{ m.}$$

则水平总力  $P_x$  为：

$$\begin{aligned} P_x &= \gamma h c A_x = 9.8 \times \frac{h}{2} \times h \times b \\ &= 9.8 \times \frac{8}{2} \times 8 \times 10 \\ &= 3136 \text{ kN} \end{aligned}$$

铅垂力为压力体 FBD 内的水体重。压力体 FBD 的体积等于面积  $S_{BDK}$  与三角形 BFD 的面积  $S_{\triangle BFD}$  之和再乘以闸孔  $b$ 。

如图中所示：  $AO = 3 \text{ m}$      $OC = 5 \text{ m}$      $OB = OD = R = 8 \text{ m}$      $FD = 8 \text{ m}$ 。

$$\text{则 } AB = \sqrt{OB^2 - AO^2} = \sqrt{8^2 - 3^2} = 7.42 \text{ m.}$$

$$CD = \sqrt{OD^2 - OC^2} = \sqrt{8^2 - 5^2} = 6.24 \text{ m}$$

$$\therefore BF = AB - CD = 7.42 - 6.24 = 1.18 \text{ m}$$

$$\therefore BD = \sqrt{BF^2 + FD^2} = \sqrt{1.18^2 + 8^2} = 8.09 \text{ m}$$

$$BE = ED = \frac{BD}{2} = \frac{8.09}{2} = 4.05 \text{ m}$$

$$\sin \alpha = \frac{BE}{OB} = \frac{4.05}{8} = 0.506 \Rightarrow \alpha = \arcsin 0.506 = 30.4^\circ$$

$$\therefore \angle BOD = 2\alpha = 60.8^\circ$$

$$\therefore S_{\text{扇形} BOD} = \frac{60.8}{360} \times \pi R^2 = \frac{60.8}{360} \times 3.14 \times 8^2 = 33.94 \text{ m}^2$$

$$OE = \sqrt{OB^2 - BE^2} = \sqrt{8^2 - 4.05^2} = 6.9 \text{ m}$$

$$S_{\triangle BOD} = \frac{1}{2} \times BD \times OE = \frac{1}{2} \times 8.09 \times 6.9 = 27.91 \text{ m}^2$$

$$\text{则 } S_{\text{BDK}} = S_{\text{扇形} BOD} - S_{\triangle BOD} = 33.94 - 27.91 = 6.03 \text{ m}^2$$

$$S_{\triangle BFD} = \frac{1}{2} \times BF \times FD = \frac{1}{2} \times 1.18 \times 8 = 4.72 \text{ m}^2$$

则压水侧面积为:

$$S_{\text{压水侧}} = S_{\text{BDK}} + S_{\triangle BFD} = 6.03 + 4.72 = 10.75 \text{ m}^2$$

$$\text{则 } V_{\text{压水侧}} = S_{\text{压水侧}} \times b = 10.75 \times 10 = 107.5 \text{ m}^3$$

$$\therefore \text{铅垂分力 } P_1 = \rho g V = 9.8 \times 107.5 = 1053.5 \text{ kN}$$

作用于闸门上的静水总压力  $P$  为

$$P = \sqrt{P_1^2 + P_2^2} = \sqrt{3136^2 + 1053.5^2} = 3308.2 \text{ kN}$$

静水总压力  $P$  的方向与水平面的夹角  $\theta$  为

$$\theta = \arctan \frac{P_2}{P_1} = \arctan \frac{1053.5}{3136} = \arctan 0.336 = 18.6^\circ$$

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2-11.

解: (1) 要使球不溢出容器, 则, 由图 2-4 (P18) 可知,

$$\frac{1}{2} \frac{\omega^2 r_0^2}{2g} \leq H-h$$

$$\text{即 } \frac{1}{2} \frac{\omega^2 \times 0.15^2}{2 \times 9.8} \leq 0.5 - 0.3 \Rightarrow \omega \leq 18.67 \text{ rad/s}$$

(2) 由图 2-4 (P18) 可知, 当水面抛物面顶点恰至角液时,

$$\text{有: } \frac{1}{2} \frac{\omega^2 r_0^2}{2g} = h$$

$$\text{即 } \frac{1}{2} \frac{\omega^2 \times 0.15^2}{2 \times 9.8} = 0.3 \Rightarrow \omega = 22.86 \text{ rad/s}$$

根据旋转抛物面的体积等于同底同高圆柱体体积的一半, 可得出当水面的抛物面顶点恰至角液时, 容器中水的体积为:

$$\begin{aligned} V &= \frac{1}{2} \pi \left(\frac{D}{2}\right)^2 H = \frac{1}{2} \times 3.14 \left(\frac{0.3}{2}\right)^2 \times 0.5 \\ &= 0.01766 \text{ m}^3 \end{aligned}$$

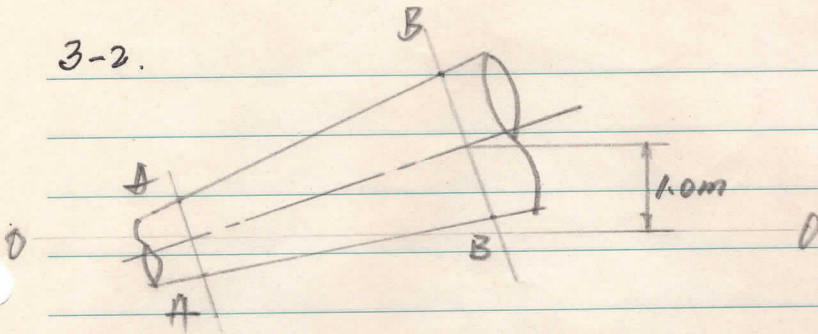
容器中原有的水体积为:

$$V' = \pi \left(\frac{D}{2}\right)^2 h = 3.14 \times \left(\frac{0.3}{2}\right)^2 \times 0.3 = 0.0212 \text{ m}^3$$

则溢出的水体积为

$$V_{\text{溢}} = V' - V = 0.0212 - 0.01766 = 0.00354 \text{ m}^3$$

3-2.



解: 如图所示.

由连续性方程可得  $V_A A_A = V_B A_B$ 即:  $V_A \pi \left(\frac{d_A}{2}\right)^2 = V_B \pi \left(\frac{d_B}{2}\right)^2$  代入数据.

$$V_A \times 3.14 \left(\frac{0.15}{2}\right)^2 = 1.5 \times 3.14 \left(\frac{0.3}{2}\right)^2 \quad \text{解出 } V_A = 6.0 \text{ m/s}$$

以通过断面 A 形心点的水平面 0-0 为基准面.

$$\text{则 } H_A = z_A + \frac{p_A}{\rho g} + \frac{\alpha V_A^2}{2g} = 0 + \frac{68.5}{9.8} + \frac{1 \times 6^2}{2 \times 9.8} = 8.83 \text{ m}$$

$$H_B = z_B + \frac{p_B}{\rho g} + \frac{\alpha V_B^2}{2g} = 1 + \frac{58}{9.8} + \frac{1 \times 1.5^2}{2 \times 9.8} = 7.03 \text{ m}$$

 $\therefore H_A > H_B$ . $\therefore$  水流方向由 A 流向 B.

则两断面的能量损失为:

$$h_{lw} = H_A - H_B = 8.83 - 7.03 = 1.8 \text{ m}$$

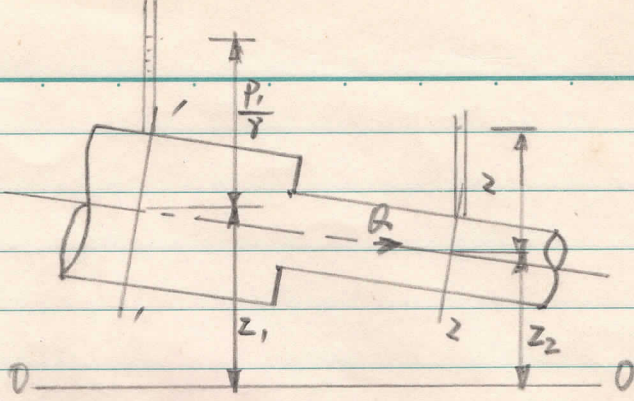
$$\text{通过管道的流量为 } Q = V_B A_B = 1.5 \times 3.14 \times \left(\frac{0.3}{2}\right)^2 = 0.106 \text{ m}^3/\text{s}$$

3-3

解:  $Q = 20 \text{ L/s} = 0.02 \text{ m}^3/\text{s}$

$$V_1 = \frac{Q}{A_1} = \frac{0.02}{0.03} = 0.667 \text{ m/s}$$

$$V_2 = \frac{Q}{A_2} = \frac{0.02}{0.01} = 2 \text{ m/s}$$



以 0-0 为基准面, 对 1-1 和 2-2 断面列能量方程

$$z_1 + \frac{p_1}{\gamma} + \frac{\alpha_1 V_1^2}{2g} = z_2 + \frac{p_2}{\gamma} + \frac{\alpha_2 V_2^2}{2g} + h_w$$

令  $\alpha_1 = \alpha_2 = 1$ , 代入数据得

$$2.5 + 1.0 + \frac{0.667^2}{2 \times 9.8} = 2 + \frac{p_2}{\gamma} + \frac{2^2}{2 \times 9.8} + 0.3 \times \frac{2^2}{2 \times 9.8}$$

$$\Rightarrow \frac{p_2}{\gamma} = 1.26 \text{ m}$$

则 测压管水头为  $z_2 + \frac{p_2}{\gamma} = 2 + 1.26 = 3.26 \text{ m}$

3-4

解: 如图示选择 1-1 和 2-2 断面

则由连续性方程得

$$V_1 A_1 = V_2 A_2$$

$$\text{即 } V_1 B H = V_2 B (H - \Delta z - \Delta h) \Rightarrow V_1 \times 2.7 \times 2 = V_2 \times 2.7 \times (2 - 0.3 - 0.2)$$

$$\Rightarrow V_1 = \frac{15}{2} V_2 \quad (1)$$

以 0-0 为基准面, 对 1-1 和 2-2 断面列能量方程如下:

$$\left. \begin{aligned} H + \frac{\alpha_1 V_1^2}{2g} &= H - \Delta h + \frac{\alpha_2 V_2^2}{2g} + h_w \\ h_w &= 0 \\ \alpha_1 &= \alpha_2 = 1 \end{aligned} \right\} \Rightarrow \frac{V_2^2 - V_1^2}{2g} = \Delta h \quad (2)$$

由 (1) 和 (2) 联立求得  $V_2 = 2.99$

$$Q = V_2 A_2 = V_2 \times (H - \Delta h - \Delta z) \times B = 2.99 \times (2 - 0.2 - 0.3) \times 2.7 = 12.11 \text{ m}^3/\text{s}$$

3-6.

解: 如图所示, 以 0-0 为基准面.

取 1-1, 2-2, 3-3, 4-4 断面.

对 1-1 和 4-4 列能量方程.

由于 c 点与大气接触,  $\therefore P_c = 0$

$$5 + \frac{\alpha_1 V_1^2}{2g} = 0 + \frac{\alpha_c V_c^2}{2g} \quad (1)$$

由于池面很大, 可知  $V_1$  很小, 即可忽略上游的流速水头.

$$\text{则式变为: } \left. \begin{aligned} 5 &= \frac{\alpha_c V_c^2}{2g} \\ \alpha_c &= 1 \end{aligned} \right\} \Rightarrow V_c = \sqrt{10g} = 9.9 \text{ m/s}$$

由于各管段管径相同, 所以  $V_A = V_B = V_c = 9.9 \text{ m/s}$ .

$$\text{则 } Q = V_c A_c = 9.9 \times \pi \times \left(\frac{0.15}{2}\right)^2 = 0.175 \text{ m}^3/\text{s}$$

对 1-1 和 2-2 断面列能量方程.

$$\left. \begin{aligned} 5 &= 5 + \frac{P_A}{\gamma} + \frac{\alpha_A V_A^2}{2g} \\ \alpha_A &= 1 \\ V_A &= 9.9 \text{ m/s} \end{aligned} \right\} \Rightarrow P_A = -49 \text{ kN/m}^2 \text{ (kPa)}$$

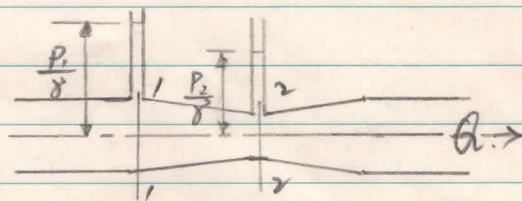
对 1-1 和 3-3 断面列能量方程.

$$\left. \begin{aligned} 5 &= 5 + 1.5 + \frac{P_B}{\gamma} + \frac{\alpha_B V_B^2}{2g} \\ \alpha_B &= 1 \\ V_B &= 9.9 \text{ m/s} \end{aligned} \right\} \Rightarrow P_B = -63.7 \text{ kN/m}^2 \text{ (kPa)}$$

若考虑能量损失, 则由 (1) 式可知,  $V_c$  要减小, 管径面积, 不变, 所以流速水头要减小, 即  $P_A$  要减小.

3-10.

解: 根据公式 (3-50), 得



$$V_2 = \frac{1}{\sqrt{1 - \left(\frac{d_2}{d_1}\right)^4}} \sqrt{2gAh}$$

$$= \frac{1}{\sqrt{1 - \left(\frac{0.1}{0.15}\right)^4}} \sqrt{2 \times 9.8 \times 0.7} = 4.135 \text{ m/s}$$

不计水头损失

理论流量

$$\text{则 } Q' = V_2 A_2 = 4.135 \times 3.14 \times \left(\frac{0.1}{2}\right)^2 = 0.0325 \text{ m}^3/\text{s} \rightarrow$$

若计水头损失, 则以过管中心线的水平面为基准面, 对 1-1、2-2 列能量方程有:

$$0 + \frac{p_1}{\gamma} + \frac{\alpha_1 V_1^2}{2g} = 0 + \frac{p_2}{\gamma} + \frac{\alpha_2 V_2^2}{2g} + h_w$$

$$\alpha_1 = \alpha_2 = 1$$

$$h_w = 0.3 \frac{V_1^2}{2g}$$

$\Rightarrow$

$$1.1 + \frac{V_1^2}{2g} = 0.4 + \frac{V_2^2}{2g} + 0.3 \frac{V_1^2}{2g} \Rightarrow 0.7 + 0.7 \frac{V_1^2}{2g} = \frac{V_2^2}{2g} \quad (1)$$

由连续性方程得:

$$V_1 A_1 = V_2 A_2 \quad \text{即} \quad V_1 \pi \left(\frac{d_1}{2}\right)^2 = V_2 \pi \left(\frac{d_2}{2}\right)^2$$

$$\Rightarrow V_1 = V_2 \left(\frac{d_2}{d_1}\right)^2 \Rightarrow V_1 = V_2 \left(\frac{0.1}{0.15}\right)^2$$

$$\Rightarrow V_1^2 = 0.1975 V_2^2 \quad (2)$$

由 (1) 和 (2) 可得  $V_2 = 3.990 \text{ m/s}$ .

$$\text{则 } Q = V_2 A_2 = 3.990 \times 3.14 \times \left(\frac{0.1}{2}\right)^2 = 0.0313 \text{ m}^3/\text{s}$$

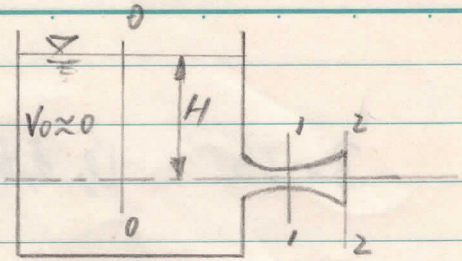
$$\text{则 } \eta = \frac{Q}{Q'} = \frac{0.0313}{0.0325} = 0.963$$

台

(CF)

3-13.

解: 如图示, 以过管嘴中心的水平面为基准面。0-0, 1-1, 2-2 分别为过水断面。



由连续性方程可知:

$$V_1 A_1 = V_2 A_2 \Rightarrow V_1 \pi \left(\frac{d_1}{2}\right)^2 = V_2 \pi \left(\frac{d_2}{2}\right)^2 \Rightarrow V_1 = \left(\frac{d_2}{d_1}\right)^2 V_2$$

$$\Rightarrow V_1 = 2.25 V_2 \quad (1)$$

对 1-1 和 2-2 断面列能量方程有:

$$0 + \frac{p_1 - p_a}{\gamma} + \frac{\alpha_1 V_1^2}{2g} = 0 + 0 + \frac{\alpha_2 V_2^2}{2g}$$

令  $\alpha_1 = \alpha_2 = 1$   
 $p_a = 101.325 \text{ kPa}$

$$\Rightarrow \frac{39.2 - 101.325}{9.8} + \frac{V_1^2}{2 \times 9.8} = \frac{V_2^2}{2 \times 9.8}$$

$$\Rightarrow -124.25 + V_1^2 = V_2^2 \quad (2)$$

由 (1) 和 (2) 联立解出  $V_2 = 5.53 \text{ m/s}$

对 0-0 和 2-2 断面列能量方程有:

$$H + 0 + 0 = 0 + 0 + \frac{V_2^2}{2g} \Rightarrow H = \frac{V_2^2}{2g} = \frac{5.53^2}{2 \times 9.8} = 1.56 \text{ m}$$

3-14.

解: 如图选 1-1 和 c-c 断面间的水体为控制体。

由连续条件可得:

$$V_1 = \frac{Q}{A_1} = \frac{37}{4.5 \times 3} = 2.741 \text{ m/s}$$

$$V_c = \frac{Q}{A_c} = \frac{37}{1.6 \times 3} = 7.708 \text{ m/s}$$

脱离体左侧受动水压力为:  $P_{左} = p_{1c} A_1 = \gamma \frac{H}{2} A_1 = 9.8 \times \frac{4.5}{2} \times 4.5 \times 3 = 297.675 \text{ kN}$

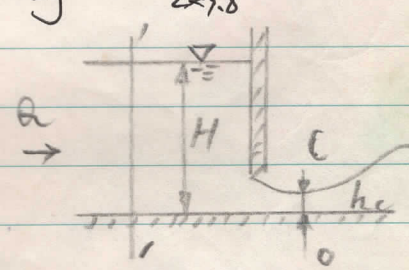
... 右侧 ...:  $P_{右} = p_{cc} A_c = \gamma \frac{h_c}{2} A_c = 9.8 \times \frac{1.6}{2} \times 1.6 \times 3 = 37.632 \text{ kN}$

以水平向右为  $x$  方向, 列动量方程:  $\rho Q (\beta_c V_c - \beta_1 V_1) = \Sigma F$

设闸门对水的力为  $T$ , 方向向左, 则

$$37 \times (7.708 - 2.741) = P_{左} - T - P_{右} = 297.675 - T - 37.632 \quad (\beta_c = \beta_1 = 1)$$

$$\Rightarrow T = 76.264 \text{ kN} \quad \text{则水对闸门的力为 } T' = 76.264 \text{ kN} \text{ 向右。} \quad (5)$$

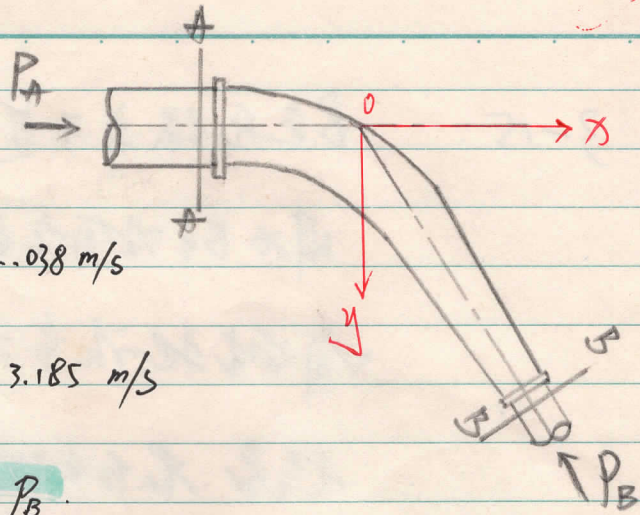


3-15.

解: 如图所示.

$$V_A = \frac{Q}{A_A} = \frac{0.1}{\pi \times (\frac{d_A}{2})^2} = \frac{0.1}{3.14 \times (\frac{0.25}{2})^2} = 2.038 \text{ m/s}$$

$$V_B = \frac{Q}{A_B} = \frac{0.1}{\pi \times (\frac{d_B}{2})^2} = \frac{0.1}{3.14 \times (\frac{0.2}{2})^2} = 3.185 \text{ m/s}$$



(1) 求 B-B 断面形心点相对压强  $P_B$ .

对 A-A 和 B-B 两断面列能量方程 (以过两断面形心点的水平面为基准面)

$$0 + \frac{P_A}{\gamma} + \frac{\alpha_A V_A^2}{2g} = 0 + \frac{P_B}{\gamma} + \frac{\alpha_B V_B^2}{2g} + 0$$

代入数据.  $\frac{150}{9.8} + \frac{2.038^2}{2 \times 9.8} = \frac{P_B}{9.8} + \frac{3.185^2}{2 \times 9.8} \Rightarrow P_B = 147.0 \text{ kN/m}^2$

(2) 求 A-A 和 B-B 断面所受的水压力  $P_A$  和  $P_B$ .

$$P_A = p_A A_A = 150 \times 3.14 \times (\frac{0.25}{2})^2 = 7.359 \text{ kN}$$

$$P_B = p_B A_B = 147 \times 3.14 \times (\frac{0.2}{2})^2 = 4.616 \text{ kN}$$

(3) 求弯管对水的力  $R_x$  和  $R_y$ .

以 A-A 和 B-B 之间的水作为脱离体, 设弯管对水的力为  $R_x$  和  $R_y$ , 方向与图中所标方向一致.

列 x 方向的动量方程.

$$\rho Q (\beta_B V_B \cos 45^\circ - \beta_A V_A) = \sum F_x = P_A + R_x - P_B \cos 45^\circ$$

$\beta_B = \beta_A = 1$ . 代入数据得.

$$0.1 \times (3.185 \times \frac{\sqrt{2}}{2} - 2.038) = 7.359 + R_x - 4.616 \times \frac{\sqrt{2}}{2}$$

$$\Rightarrow R_x = -4.074 \text{ kN}$$

说明弯管对水的力沿 x 方向上的力与 x 方向相反. 大小为

4.074 kN

列沿方向的平衡方程

$$P_A(\beta_B V_B \sin 45^\circ - 0) = \sum F_y = R_y - P_B \sin 45^\circ$$

$\beta_B = \beta_A = 1$ . 代入数据得

$$0.1 \times (3.185 \times \frac{\sqrt{2}}{2} - 0) = R_y - 4.616 \times \frac{\sqrt{2}}{2}$$

$$\Rightarrow R_y = 3.489 \text{ kN}$$

说明弯管对水体沿方向上的力与方向一致, 大小为 3.489 kN.

(4) 求水流对弯管的作用力  $R_x'$  和  $R_y'$

经分析可知水流对弯管的作用力和弯管对水流的作用力是作用力与反作用力的关系, 所以有:

$$R_x' = -R_x = 4.074 \text{ kN}$$

$$R_y' = -R_y = -3.489 \text{ kN}$$

$$\text{则 } R' = \sqrt{R_x'^2 + R_y'^2} = \sqrt{4.074^2 + (-3.489)^2} = 5.364 \text{ kN}$$

故  $R'$  与 x 轴的夹角为

$$\alpha = \arctan \frac{R_y'}{R_x'} = \arctan \frac{3.489}{4.074} = 40.58^\circ$$



4-1.

解:  $d = 0.015 \text{ m}$      $t = 12^\circ \text{C} \rightarrow \nu = 1.239 \times 10^{-6} \text{ m}^2/\text{s}$  (查表 1-1)

$v = 0.15 \text{ m/s}$

则  $Re = \frac{vd}{\nu} = \frac{0.15 \times 0.015}{1.239 \times 10^{-6}} = 1815.98 < 2000$

∴ 水流是层流.

4-5.

解:  $d = 0.25 \text{ m}$      $\Delta = 0.0005 \text{ m}$      $t = 10^\circ \text{C} \rightarrow \nu = 1.306 \times 10^{-6} \text{ m}^2/\text{s}$

根据书 P<sub>107</sub> 可知, 当  $\frac{\Delta}{\delta_0} > 6$  时是粗糙区.

所以, 流量最小时对应的粗糙区应满足  $\frac{\Delta}{\delta_0} = 6$

要计算  $\delta_0$  需先计算粘性底层厚度  $\delta_0$  (可按 4-32 公式计算).

要计算  $\delta_0$  需先求出  $\lambda$ .

则, 可用 尼古拉兹公式  $\lambda = \frac{1}{[2.3 \lg(3.7 \frac{d}{\lambda})]^2}$  (4-46) 计算.

$\lambda = \frac{1}{[2.3 \lg(3.7 \frac{0.25}{\lambda})]^2} = 0.02342$

∴ 水断面面积,  $A = \pi (\frac{d}{2})^2 = 3.14 \times (\frac{0.25}{2})^2 = 0.04906 \text{ m}^2$ .

$v = \frac{Q_{min}}{0.04906} \text{ m/s}$

则  $\delta_0 = \frac{32.8d}{Re \sqrt{\lambda}} = \frac{32.8 \times 0.25}{\frac{vd}{\nu} \sqrt{\lambda}} = \frac{32.8 \times 0.25}{\frac{Q_{min}}{0.04906} \times 0.25 \times \sqrt{0.02342}} \times \frac{1.306 \times 10^{-6}}{1.306 \times 10^{-6}}$

化简.  $\delta_0 = \frac{13.732 \times 10^{-6}}{Q_{min}}$

则  $\frac{\Delta}{\delta_0} = 6 \Rightarrow \frac{0.0005}{\frac{13.732 \times 10^{-6}}{Q_{min}}} = 6 \Rightarrow Q_{min} = 0.165 \text{ m}^3/\text{s}$

合数

(1)

$$\left. \begin{aligned} \tau_0 &= \gamma R J && (4-13) \text{ P94} \\ R &= \frac{d}{4} && \text{P90-95} \\ J &= \frac{h_f}{L} = \frac{\lambda \frac{L}{d} \frac{v^2}{2g}}{L} = \frac{\lambda}{d} \frac{v^2}{2g} \end{aligned} \right\} \Rightarrow \tau_0 = \gamma \frac{d}{4} \times \frac{\lambda}{d} \frac{v^2}{2g} = \frac{\gamma \lambda v^2}{8g}$$

$$v = \frac{Q_{\min}}{A} = \frac{0.165}{0.04906} = 3.36 \text{ m/s}$$

$$\text{则} \tau_0 = \frac{\gamma \lambda v^2}{8g} = \frac{9.8 \times 0.02342 \times 3.36^2}{8 \times 9.8} = 0.132 \text{ N/m}^2$$

4-7.

$$\text{解: } d = 0.05 \text{ m}, \quad h_f = 0.8 \text{ m}$$

$$L = 10 \text{ m}$$

由已知条件可得:

$$Q = \frac{V}{t} = \frac{0.247}{90} = 0.00274 \text{ m}^3/\text{s}$$

$$\text{则} v = \frac{Q}{A} = \frac{0.00274}{3.14 \times (\frac{0.05}{2})^2} = 1.396 \text{ m/s}$$

$$h_f = 0.8 = \lambda \frac{L}{d} \frac{v^2}{2g} \Rightarrow 0.8 = \lambda \frac{10}{0.05} \times \frac{1.396^2}{2 \times 9.8} \Rightarrow \lambda = 0.0402$$

4-10.

$$\text{解: } l = 10 \text{ m}, \quad d = 0.05 \text{ m}, \quad \lambda = 0.03$$

$$h_w = 0.629$$

由已知条件可得:

$$Q = \frac{V}{t} = \frac{0.329}{2 \times 60} = 0.00274 \text{ m}^3/\text{s}$$

$$v = \frac{Q}{A} = \frac{0.00274}{3.14 \times (\frac{0.05}{2})^2} = 1.396 \text{ m/s} \quad h_f = \lambda \frac{L}{d} \frac{v^2}{2g} = 0.03 \times \frac{10}{0.05} \times \frac{1.396^2}{2 \times 9.8} = 0.597 \text{ m}$$

$$h_w = h_f + h_j \quad \text{即} \quad h_w = h_f + \xi \frac{v^2}{2g}$$

$$\text{代入数据} \quad 0.629 = 0.597 + \xi \frac{1.396^2}{2 \times 9.8} \Rightarrow \xi = 0.322$$

答

(2)

4-12.

查表 1-1

$$\text{解: } l=300\text{ m}, d=5\text{ m}, t=20^\circ\text{C} \rightarrow \nu = 1.003 \times 10^{-6} \text{ m}^2/\text{s}$$

$$Q=200 \text{ m}^3/\text{s}$$

$$V = \frac{Q}{A} = \frac{200}{3.14 \times (\frac{5}{2})^2} = 10.19 \text{ m/s}$$

$$Re = \frac{vd}{\nu} = \frac{10.19 \times 5}{1.003 \times 10^{-6}} = 5.08 \times 10^7$$

用公式 (4-50) P112  
P107

查表 4-2 P113  $k_s = 0.3 \sim 0.9 \text{ mm}$ . 取  $k_s = 0.5 \text{ mm}$

用公式 (4-50) 计算  $\lambda$ .

当量粗糙度

$$\lambda = \frac{1.325}{\left[ \ln \left( \frac{\Delta}{3.7d} + \frac{5.74}{Re^{0.9}} \right) \right]^2} = \frac{1.325}{\left[ \ln \left( \frac{0.5 \times 10^{-3}}{3.7 \times 5} + \frac{5.74}{(5.08 \times 10^7)^{0.9}} \right) \right]^2} = 0.01203$$

(1). 用达西-威斯巴赫公式计算  $h_f$ .

$$h_f = \lambda \frac{l}{d} \frac{V^2}{2g} = 0.01203 \times \frac{300}{5} \times \frac{10.19^2}{2 \times 9.8} = 3.824 \text{ m}$$

(2). 用谢才公式计算  $h_f$

$$C = \frac{1}{n} R^{\frac{1}{6}} \quad \text{查表 4-5 } n = 0.012 \sim 0.014 \quad \text{取 } n = 0.013$$

$$R = \frac{d}{4} = \frac{5}{4}$$

$$\text{则 } C = \frac{1}{0.013} \left( \frac{5}{4} \right)^{\frac{1}{6}} = 79.838$$

$$V = C \sqrt{RJ} \Rightarrow 10.19 = 79.838 \sqrt{\frac{5}{4} J} \Rightarrow J = 0.01303$$

$$\text{则 } h_f = J l = 0.01303 \times 300 = 3.909 \text{ m}$$

## 第五章 有压管道中的恒定流

### 习题 5-1

解:

根据题意可知, 此为简单管道自由出流, 所用公式为,

$$Q = \mu_c A \sqrt{2gH_0}$$

其中,

$$\begin{aligned}\mu_c &= \frac{1}{\sqrt{1 + \lambda \frac{l}{d} + \sum \zeta}} = \frac{1}{\sqrt{1 + 0.025 \times \frac{800}{0.15} + \zeta_{\text{进口}} + 2 \times \zeta_{\text{弯头}}}} \\ &= \frac{1}{\sqrt{1 + 0.025 \times \frac{800}{0.15} + 0.5 + 2 \times 0.3}} \\ &= 0.086\end{aligned}$$

通过管道的流量为,

$$Q = \mu_c A \sqrt{2gH_0} = 0.086 \times 3.14 \times \frac{0.15^2}{4} \times \sqrt{2 \times 9.8 \times 20} = 0.03 \text{ m}^3/\text{s}$$

### 习题 5-3

解:

倒虹吸管属于有压管道淹没出流。若不考虑上下游渠道中行近流速, 则可直接应用淹没出流的公式  $Q = \mu_c A \sqrt{2gz}$  计算流量。

根据题意可知:  $z = 0.5m$

铸铁管道沿程阻力系数  $\lambda$  的计算, 先假定管道在阻力平方区工作, 再用曼宁公式计算谢才系数  $C$ , 然后利用沿程阻力系数和谢才系数的关系确定  $\lambda$ 。

铸铁管,  $n=0.01 \sim 0.014$ , 本题取  $n=0.013$

$$C = \frac{1}{n} R^{\frac{1}{6}} = \frac{1}{0.013} \left(\frac{0.5}{4}\right)^{\frac{1}{6}} = 54.39$$

$$\lambda = \frac{8g}{C^2} = \frac{8 \times 9.8}{54.39^2} = 0.0265$$

管道流量系数为:

$$\mu_c = \frac{1}{\sqrt{\lambda \frac{l}{d} + \zeta_{\text{进口}} + \zeta_{60^\circ} + \zeta_{50^\circ} + \zeta_{\text{出口}}}} = \frac{1}{\sqrt{0.0265 \times \frac{125}{0.5} + 0.5 + 0.55 + 0.4 + 1}} = 0.332$$

则倒虹吸管通过的流量为:

$$Q = \mu_c A \sqrt{2gz} = 0.332 \times 3.14 \times \left(\frac{0.5}{2}\right)^2 \times \sqrt{2 \times 9.8 \times 0.5} = 0.204 m^3/s$$

倒虹吸管的流速为: 
$$v = \frac{Q}{A} = \frac{0.204}{3.14 \times \left(\frac{0.5}{2}\right)^2} = 1.04 m/s$$

流速较大, 可知管流属于阻力平方区, 满足谢才公式应用条件, 前面假定是合理的。

5-5.

解: 水泵扬程的计算为:

$$h_p = H + h_{w吸} + h_{w压}$$

$$H = 179.5 - 155.0 = 24.5 \text{ m}$$

$$h_w = \left( \lambda \frac{l}{d} + \sum \xi \right) \frac{v^2}{2g}$$

$\lambda$  计算, 按阻力平方区计算

$$\text{即 } \lambda = \frac{8g}{c^2} \quad c = \frac{1}{n} R^{\frac{1}{6}}$$

钢管  $n = 0.011 \sim 0.0125$   $\rightarrow$  取  $n = 0.012$

(1). 吸水计算.

$$V_{吸} = \frac{0.05}{3.14 \times \left(\frac{0.2}{2}\right)^2} = 1.592 \text{ m/s}$$

查表 4-6.  $45^\circ$  弯管.  $\xi_1 = 0.35$ ; 带阀门的逆弯头  $\xi_2 = 5.2$   
( $d = 200 \text{ mm}$ )

$$c = \frac{1}{n} R^{\frac{1}{6}} = \frac{1}{n} \left(\frac{d_{吸}}{4}\right)^{\frac{1}{6}} = \frac{1}{0.012} \times \left(\frac{0.2}{4}\right)^{\frac{1}{6}} = 50.58$$

$$\lambda = \frac{8g}{c^2} = \frac{8 \times 9.8}{50.58^2} = 0.0078 \quad 0.0306$$

$$h_{w吸} = \left( \lambda \frac{l}{d} + \sum \xi \right) \frac{v^2}{2g} = \left( 0.0078 \times \frac{4}{0.2} + 5.2 + 0.35 \right) \times \frac{1.592^2}{2 \times 9.8} = 0.797 \text{ m}$$

(2). 压水计算.

$$V_{压} = \frac{0.05}{3.14 \times \left(\frac{0.15}{2}\right)^2} = 2.831 \text{ m/s}$$

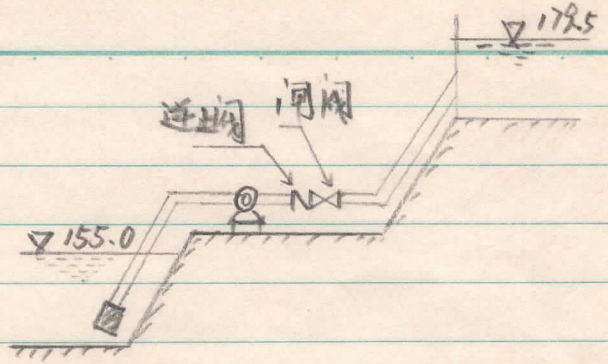
$$c = \frac{1}{n} R^{\frac{1}{6}} = \frac{1}{n} \left(\frac{d_{压}}{4}\right)^{\frac{1}{6}} = \frac{1}{0.012} \times \left(\frac{0.15}{4}\right)^{\frac{1}{6}} = 48.21$$

$$\lambda = \frac{8g}{c^2} = \frac{8 \times 9.8}{48.21^2} = 0.0337$$

$$h_{w压} = \left( \lambda \frac{l}{d} + \sum \xi \right) \frac{v^2}{2g} = \left( 0.0337 \times \frac{50}{0.15} + 1.7 + 0.1 + 0.35 + 1 \right) \times \frac{2.831^2}{2 \times 9.8} = 5.881 \text{ m}$$

$$\therefore h_p = H + h_{w吸} + h_{w压} = 24.5 + 0.797 + 5.881 = 31.178 \text{ m}$$

(2)



### 习题 5-8

解:

设管段  $AB$ 、 $BC$ 、 $CD$  的流量分别为  $Q_1$ 、 $Q_2$ 、 $Q_3$ ，流量模数分别为  $K_1$ 、 $K_2$ 、 $K_3$ ，速度分别为  $v_1$ 、 $v_2$ 、 $v_3$ ，该管路系统为串联管道，各管段均为简单管道，不计局部水头损失，则总水头损失为：

$$H = \frac{Q_1^2}{K_1^2} l_1 + \frac{Q_2^2}{K_2^2} l_2 + \frac{Q_3^2}{K_3^2} l_3$$

根据连续性方程可得：

$$Q_3 = q_D = 0.01 \text{ m}^3/\text{s}; \quad Q_2 = Q_3 + q_C = 0.015 \text{ m}^3/\text{s}; \quad Q_1 = Q_2 + q_B = 0.025 \text{ m}^3/\text{s}$$

$$v_1 = \frac{4Q_1}{\pi d_1^2} = \frac{4 \times 0.025}{3.14 \times 0.2^2} = 0.796 \text{ m/s}$$

$$v_3 = \frac{4Q_3}{\pi d_3^2} = \frac{4 \times 0.01}{3.14 \times 0.1^2} = 1.274 \text{ m/s}$$

铸铁管， $n=0.01 \sim 0.014$ ，本题取  $n=0.013$

则各管段谢才系数分别为：

$$C_1 = \frac{1}{n} R_1^{\frac{1}{6}} = \frac{1}{n} \left( \frac{d_1}{4} \right)^{\frac{1}{6}} = \frac{1}{0.013} \times \left( \frac{0.2}{4} \right)^{\frac{1}{6}} = 46.689$$

$$C_2 = \frac{1}{n} R_2^{\frac{1}{6}} = \frac{1}{n} \left( \frac{d_2}{4} \right)^{\frac{1}{6}} = \frac{1}{0.013} \times \left( \frac{0.15}{4} \right)^{\frac{1}{6}} = 44.504$$

$$C_3 = \frac{1}{n} R_3^{\frac{1}{6}} = \frac{1}{n} \left( \frac{d_3}{4} \right)^{\frac{1}{6}} = \frac{1}{0.013} \times \left( \frac{0.1}{4} \right)^{\frac{1}{6}} = 41.596$$

则各管段流量模数分别为：

$$K_1 = A_1 C_1 \sqrt{R_1} = \frac{\pi d_1^2}{4} \times C_1 \times \sqrt{\frac{d_1}{4}} = \frac{3.14 \times 0.2^2}{4} \times 46.689 \times \sqrt{\frac{0.2}{4}} = 0.328 \text{ m}^3/\text{s}$$

$$K_2 = A_2 C_2 \sqrt{R_2} = \frac{\pi d_2^2}{4} \times C_2 \times \sqrt{\frac{d_2}{4}} = \frac{3.14 \times 0.15^2}{4} \times 44.504 \times \sqrt{\frac{0.15}{4}} = 0.152 \text{ m}^3/\text{s}$$

$$K_3 = A_3 C_3 \sqrt{R_3} = \frac{\pi d_3^2}{4} \times C_3 \times \sqrt{\frac{d_3}{4}} = \frac{3.14 \times 0.1^2}{4} \times 41.596 \times \sqrt{\frac{0.1}{4}} = 0.052 \text{ m}^3/\text{s}$$

总水头损失为：

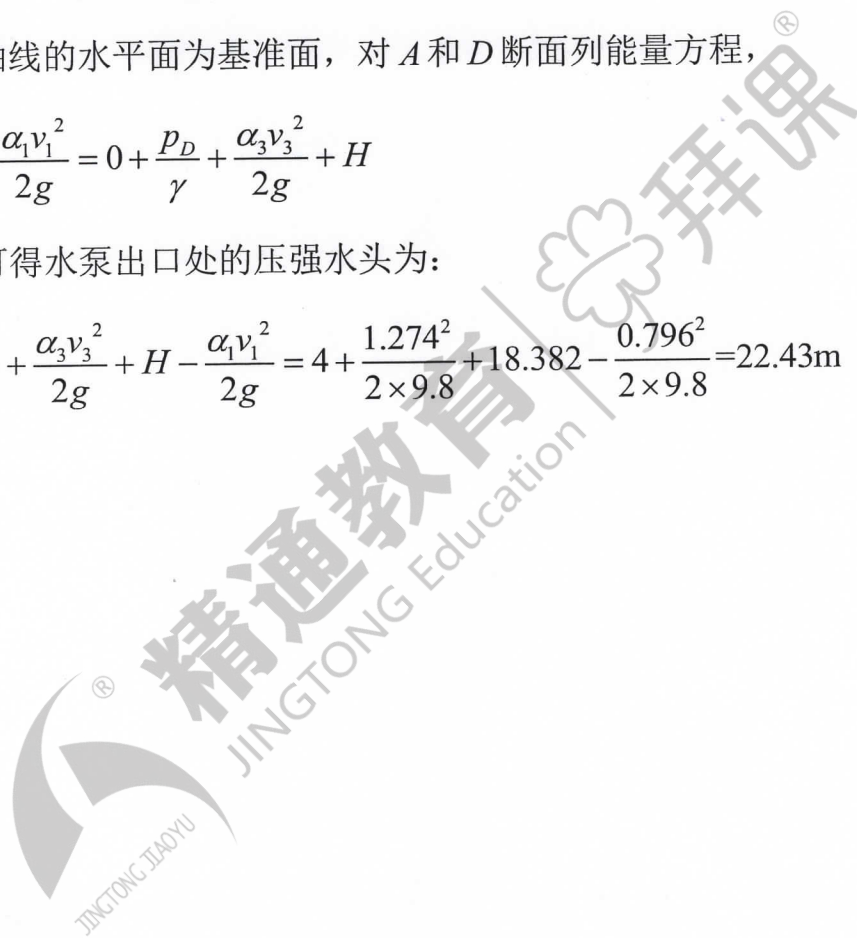
$$H = \frac{Q_1^2}{K_1^2} l_1 + \frac{Q_2^2}{K_2^2} l_2 + \frac{Q_3^2}{K_3^2} l_3 = \frac{0.025^2}{0.328^2} \times 500 + \frac{0.015^2}{0.152^2} \times 450 + \frac{0.01^2}{0.052^2} \times 300 = 18.382 \text{ m}$$

以过管轴线的水平面为基准面，对  $A$  和  $D$  断面列能量方程，

$$0 + \frac{p_A}{\gamma} + \frac{\alpha_1 v_1^2}{2g} = 0 + \frac{p_D}{\gamma} + \frac{\alpha_3 v_3^2}{2g} + H$$

从上式可得水泵出口处的压强水头为：

$$\frac{p_A}{\gamma} = \frac{p_D}{\gamma} + \frac{\alpha_3 v_3^2}{2g} + H - \frac{\alpha_1 v_1^2}{2g} = 4 + \frac{1.274^2}{2 \times 9.8} + 18.382 - \frac{0.796^2}{2 \times 9.8} = 22.43 \text{ m}$$





## 第六章 明渠恒定均匀流

### 习题 6-2

解:

$$(b+mk)h = (2.4 + 1.5 \times 1.2) \times 1.2 = 5.04 \text{ m}^2$$

过水断面面积  $A = \cancel{(b+mk)h} = \cancel{(2.4 + 1.5 \times 1.2) \times 1.2} = 5.04$

湿周  $\chi = b + 2h\sqrt{1+m^2} = 2.4 + 2 \times 1.2 \times \sqrt{1+1.5^2} = 6.727 \text{ m}$

水力半径  $R = \frac{A}{\chi} = \frac{5.04}{6.727} = 0.749$

$$Q = AC\sqrt{Ri} = A \times \frac{1}{n} R^{\frac{1}{6}} \sqrt{Ri} = 5.04 \times \frac{1}{0.025} \times 0.749^{\frac{1}{6}} \sqrt{0.749 \times 0.0016} = 6.65 \text{ m}^3/\text{s}$$

$$v = \frac{Q}{A} = \frac{6.65}{5.04} = 1.319 \text{ m/s}$$

### 习题 6-4

解:

由于是按照水力最优断面设计, 所以有

$$\beta_m = \frac{b}{h} = 2$$

则,  $h = \frac{b}{2} = \frac{8}{2} = 4 \text{ m}$ ,  $A = bh = 8 \times 4 = 32 \text{ m}^2$ ,  $R = \frac{h}{2} = \frac{4}{2} = 2 \text{ m}$

$$Q = AC\sqrt{Ri} = A \times \frac{1}{n} R^{\frac{2}{3}} \sqrt{i} = 32 \times \frac{1}{0.028} \times 2^{\frac{2}{3}} \sqrt{\frac{1}{8000}} = 22.283 \text{ m}^3/\text{s}$$

### 习题 6-7

解:

过水断面面积  $A = \cancel{bh} = \cancel{1.5 \times 1.7} = \cancel{2.55} \text{ m}^2$   
 $A = bh = 1.5 \times 1.7 = 2.55 \text{ m}^2$

$$\text{湿周} \quad \chi = b + 2h = 1.5 + 2 \times 1.7 = 4.9 \text{ m}$$

$$\text{水力半径} \quad R = \frac{A}{\chi} = \frac{2.55}{4.9} = 0.52$$

槽身壁面为净水泥抹面,  $n=0.01 \sim 0.013$ , 本题取  $n=0.011$

$$C = \frac{1}{n} R^{\frac{1}{6}} = \frac{1}{0.011} \times (0.52)^{\frac{1}{6}} = 81.52$$

$$Q = AC\sqrt{Ri}$$

$$\text{则 } i = \frac{Q^2}{A^2 C^2 R} = \frac{7.65^2}{2.55^2 \times 81.52^2 \times 0.52} = 0.0026$$

设渡槽出口处槽底高程为  $z_{\text{出口}}$

$$i = \tan \theta = \frac{\Delta Z}{\Delta S}$$

$$\text{上式带入数据得到 } 0.0026 = \frac{52.06 - z_{\text{出口}}}{116.5}$$

$$\text{则渡槽出口处槽底高程为 } z_{\text{出口}} = 52.06 - 116.5 \times 0.0026 = 51.76 \text{ m}$$

### 习题 6-14

解:

(1)

$$\text{用试算法求解水深, 先计算流量模数 } K = \frac{Q}{\sqrt{i}} = \frac{6.7}{\sqrt{\frac{1}{6500}}} = 540.17 \text{ m}^3/\text{s}$$

假设水深  $h = 1.0 \text{ m}$

$$\text{过水断面面积} \quad A = (b + mh)h = (3.4 + 1.5 \times 1) \times 1 = 4.9 \text{ m}^2$$

$$\text{湿周} \quad \chi = b + 2h\sqrt{1 + m^2} = 3.4 + 2 \times 1 \times \sqrt{1 + 1.5^2} = 7.006 \text{ m}$$

水力半径  $R = \frac{A}{\chi} = \frac{4.9}{7.006} = 0.699$

谢才系数  $C = \frac{1}{n} R^{\frac{1}{6}} = \frac{1}{0.03} \times (0.699)^{\frac{1}{6}} = 31.405$

流量模数  $K' = AC\sqrt{R} = 4.9 \times 31.405 \times \sqrt{0.699} = 128.698 \text{ m}^3/\text{s}$

因为  $K' \neq K$ ，且差别较大，所以假设水深  $h = 1.0\text{m}$  不正确，重新假设水深再计算，直到  $K'$  与  $K$  接近相等，则假设的水深  $h$  即为所求。

如下表所示，分别假设水深为 1m、1.5m、2.1m、2.114m 和 2.2m，则当水深  $h = 2.114\text{m}$ ， $K'$  与  $K$  非常接近相等，所以所求水深为  $h = 2.114\text{m}$ 。

$h$	1	1.5	2.1	2.114	2.2
$K'$	128.698	264.844	533.091	540.255	585.466

(2)

保证超高 0.5m，则水深  $h = 3.2 - 0.5 = 2.7\text{m}$ ，

过水断面面积  $A = (b + mh)h = (3 + 4 \times 1.5) \times 2.7 = 20.115 \text{ m}^2$

湿周  $\chi = b + 2h\sqrt{1 + m^2} = 3.4 + 2 \times 2.7 \times \sqrt{1 + 1.5^2} = 13.135\text{m}$

水力半径  $R = \frac{A}{\chi} = \frac{20.115}{13.135} = 1.531$

谢才系数  $C = \frac{1}{n} R^{\frac{1}{6}} = \frac{1}{0.03} \times (1.531)^{\frac{1}{6}} = 35.788$

通过流量  $Q = AC\sqrt{Ri} = 20.115 \times 35.788 \times \sqrt{1.531 \times \frac{1}{6500}} = 11.049 \text{ m}^3/\text{s}$

尚能供给工业用水流量为：

$Q_{\text{工业}} = 11.049 - 6.7 = 4.349 \text{ m}^3/\text{s}$

此时渠道中流速为：

$$v = \frac{Q}{A} = \frac{11.049}{20.115} = 0.549 \text{ m/s}$$

根据表 6-5 可知，黏土渠道不冲允许流速为 0.75~0.95m/s，渠道中流速  $v = 0.549 \text{ m/s} < 0.75 \text{ m/s}$ ，所以渠道中不会发生冲刷。

(3)

当水深  $h = 1.5 \text{ m}$ ，

$$\text{过水断面面积 } A = (b + mh)h = (3 + 1.5 \times 1.5) \times 1.5 = 8.475 \text{ m}^2$$

$$\text{湿周 } \chi = b + 2h\sqrt{1 + m^2} = 3.4 + 2 \times 1.5 \times \sqrt{1 + 1.5^2} = 8.808 \text{ m}$$

$$\text{水力半径 } R = \frac{A}{\chi} = \frac{8.475}{8.808} = 0.962$$

$$\text{谢才系数 } C = \frac{1}{n} R^{\frac{1}{6}} = \frac{1}{0.03} \times (0.962)^{\frac{1}{6}} = 33.120$$

$$\text{通过流量 } Q = AC\sqrt{Ri} = 8.475 \times 33.120 \times \sqrt{0.962 \times \frac{1}{6500}} = 3.415 \text{ m}^3/\text{s}$$

此时渠道中流速为：

$$v = \frac{Q}{A} = \frac{3.415}{8.475} = 0.403 \text{ m/s} < v_{\text{不淤}} = 0.5 \text{ m/s}$$

所以渠道中会发生淤积现象。

(4)

如下表所示，当水深分别取不同值时，求出对应流量，绘制水深与流量关系曲线如下图所示。

$h$ (m)	1	1.5	2	2.114	2.7	3
$Q$ (m <sup>3</sup> /s)	1.596	3.415	5.997	6.7	11.049	13.777

## 第七章 明渠恒定非均匀流

### 习题 7-1

解:

(1)

用试算法求解正常水深, 先计算流量模数  $K = \frac{Q}{\sqrt{i}} = \frac{4}{\sqrt{0.0005}} = 178.885 \text{ m}^3/\text{s}$

假设水深  $h = 1.0 \text{ m}$

过水断面面积  $A = (b + mh)h = (2.5 + 0 \times 1) \times 1 = 2.5 \text{ m}^2$

湿周  $\chi = b + 2h\sqrt{1 + m^2} = 2.5 + 2 \times 1 \times \sqrt{1 + 0^2} = 4.5 \text{ m}$

水力半径  $R = \frac{A}{\chi} = \frac{2.5}{4.5} = 0.556$

谢才系数  $C = \frac{1}{n} R^{\frac{1}{6}} = \frac{1}{0.02} \times (0.556)^{\frac{1}{6}} = 45.333$

流量模数  $K' = AC\sqrt{R} = 2.5 \times 45.333 \times \sqrt{0.556} = 84.473 \text{ m}^3/\text{s}$

因为  $K' \neq K$ , 且差别较大, 所以假设水深  $h = 1.0 \text{ m}$  不正确, 重新假设水深再计算, 直到  $K'$  与  $K$  接近相等, 则假设的水深  $h$  即为所求。

如下表所示, 分别假设水深为 1m、1.5m、2.1m、2.114m 和 2.2m, 则当水深  $h = 1.762 \text{ m}$ ,  $K'$  与  $K$  非常接近相等, 所以所求水深为  $h = 1.762 \text{ m}$ 。

$h$	1	1.6	1.762	1.8
$K'$	84.473	157.936	178.766	183.696

则微波波速  $\omega = \sqrt{gh} = \sqrt{9.8 \times 1.762} = 4.155 \text{ m/s}$

(2)

当水深  $h = 1.762 \text{ m}$  时, 则断面面积为:

$$A = bh = 2.5 \times 1.762 = 4.405 \text{ m}^2$$

此时渠道中流速为： $v = \frac{Q}{A} = \frac{4}{4.405} = 0.908\text{m/s}$

$$F_r = \frac{v}{\omega} = \frac{0.908}{4.155} = 0.219 < 1$$

渠道中水流为缓流。

### 习题 7-3

解：

(1) 对于圆形无压隧洞，临界水深的求解可依据临界流方程  $\frac{\alpha Q^2}{g} = \frac{A_k^3}{B_k}$ ，利用试

算法求得：

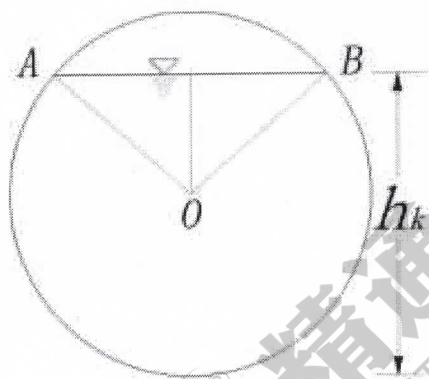


图 1

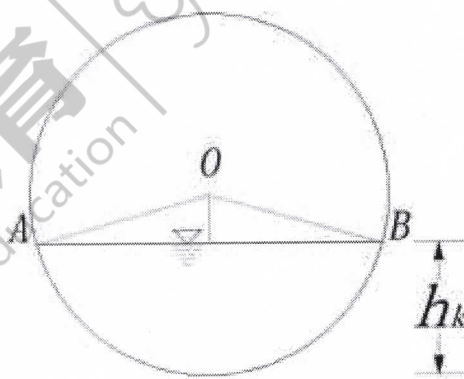


图 2

先计算方程左侧值  $\frac{\alpha Q^2}{g} = \frac{10.5^2}{9.8} = 11.25$

假设临界水深超过隧洞半径，如上图 1 所示，则临界水深所对应的过水断面面积：

$$A_k = \pi(d/2)^2 - \left[ \frac{2 \arccos\left(\frac{h_k - d/2}{d/2}\right)}{2\pi} \pi(d/2)^2 - 2 \times \sqrt{d/2^2 - (h_k - d/2)^2} \times (h_k - d/2) \times \frac{1}{2} \right]$$

$$\text{化简后: } A_k = 4.906 - \left( 1.5625 \times \arccos \left( \frac{h_k - 1.25}{1.25} \right) - \sqrt{2.5h_k - h_k^2} \times (h_k - 1.25) \right)$$

$$B_k = 2 \times \sqrt{\left( \frac{d}{2} \right)^2 - \left( h_k - \frac{d}{2} \right)^2} = 2 \times \sqrt{2.5h_k - h_k^2}$$

假设水深  $h_k = 1.3\text{m}$

$$\text{计算 } \frac{A_k^3}{B_k} = 6.58 \neq \frac{\alpha Q^2}{g} = 11.25, \text{ 所以临界水深 } h_k \neq 1.3\text{m}$$

...

继续假设水深  $h_k = 1.48\text{m}$

$$\text{计算 } \frac{A_k^3}{B_k} = 11.246 \text{ 与 } \frac{\alpha Q^2}{g} = 11.25 \text{ 非常接近, 所以临界水深为 } h_k = 1.48\text{m}$$

假设临界水深小于隧洞半径, 如上图 2 所示, 则临界水深所对应的过水断面面积:

$$A_k = \frac{2 \arccos \left( \frac{d/2 - h_k}{d/2} \right)}{2\pi} \pi (d/2)^2 - 2 \times \sqrt{d/2^2 - d/2 - h_k^2} \times (d/2 - h_k) \times \frac{1}{2}$$

$$\text{化简后: } A_k = 1.5625 \times \arccos \left( \frac{1.25 - h_k}{1.25} \right) - \sqrt{2.5h_k - h_k^2} \times (1.25 - h_k)$$

$$B_k = 2 \times \sqrt{\left( \frac{d}{2} \right)^2 - \left( \frac{d}{2} - h_k \right)^2} = 2 \times \sqrt{2.5h_k - h_k^2}$$

通过假设临界水深, 发现当临界水深小于隧洞半径时, 没有解。

$$\text{即 } \frac{A_k^3}{B_k} \neq \frac{\alpha Q^2}{g}, \text{ 所以临界水深应该大于半径。}$$

(2) 根据上步所求临界水深, 可求出临界流时断面面积为:

$$\begin{aligned}
 A_k &= 4.906 - \left( 1.5625 \times \arccos \left( \frac{h_k - 1.25}{1.25} \right) - \sqrt{2.5h_k - h_k^2} \times (h_k - 1.25) \right) \\
 &= 4.906 - \left( 1.5625 \times \arccos \left( \frac{1.48 - 1.25}{1.25} \right) - \sqrt{2.5 \times 1.48 - 1.48^2} \times (1.48 - 1.25) \right) \\
 &= 3.023 \text{m}^2
 \end{aligned}$$

临界流流速为:  $v_k = \frac{Q}{A_k} = \frac{10.5}{3.023} = 3.473 \text{m/s}$

(3) 当均匀流水深  $h_0 = 2.1 \text{m}$  时, 根据 (1) 中可知此时过水断面面积为:

$$\begin{aligned}
 A &= 4.906 - \left( 1.5625 \times \arccos \left( \frac{h_0 - 1.25}{1.25} \right) - \sqrt{2.5h_0 - h_0^2} \times (h_0 - 1.25) \right) \\
 &= 4.906 - \left( 1.5625 \times \arccos \left( \frac{2.1 - 1.25}{1.25} \right) - \sqrt{2.5 \times 2.1 - 2.1^2} \times (2.1 - 1.25) \right) \\
 &= 4.399 \text{m}^2
 \end{aligned}$$

水面宽度为:

$$B = 2 \times \sqrt{2.5h_k - h_k^2} = 2 \times \sqrt{2.5 \times 2.1 - 2.1^2} = 1.833 \text{m}$$

此时平均水深为:  $\bar{h} = \frac{A}{B} = \frac{4.399}{1.833} = 2.4 \text{m}$

$$\omega = \sqrt{g\bar{h}} = \sqrt{9.8 \times 2.4} = 4.85 \text{m/s}$$

(4) 解:

$h_k = 1.48 \text{m}$ ,  $h_0 = 2.1 \text{m}$ ,  $h_0 = 2.1 \text{m} > h_k = 1.48 \text{m}$ , 所以均匀流时的流态为缓流。

### 习题 7-6

解:

(1) 求正常水深

用试算法求解正常水深, 先计算流量模数  $K = \frac{Q}{\sqrt{i}} = \frac{20}{\sqrt{0.0016}} = 500 \text{m}^3/\text{s}$



假设水深  $h_0 = 1.0\text{m}$

过水断面面积  $A = (b + mh_0)h_0 = (3 + 1 \times 1) \times 1 = 4\text{m}^2$

湿周  $\chi = b + 2h_0\sqrt{1 + m^2} = 3 + 2 \times 1 \times \sqrt{1 + 1^2} = 5.828\text{m}$

水力半径  $R = \frac{A}{\chi} = \frac{4}{5.828} = 0.686$

谢才系数  $C = \frac{1}{n} R^{\frac{1}{6}} = \frac{1}{0.015} \times (0.686)^{\frac{1}{6}} = 62.612$

流量模数  $K' = AC\sqrt{R} = 4 \times 62.612 \times \sqrt{0.686} = 207.477\text{m}^3/\text{s}$

因为  $K' \neq K$ ，且差别较大，所以假设正常水深  $h_0 = 1.0\text{m}$  不正确，重新假设水深再计算，直到  $K'$  与  $K$  接近相等，则假设的水深  $h$  即为所求。

$h_0$	1	1.5	1.7	1.6	1.629
$K'$	207.477	429.355	540.898	483.477	499.759

如上表所示，分别假设正常水深为 1m、1.5m、1.7m、1.6m 和 1.629m，则当正常水深  $h_0 = 1.629\text{m}$ ， $K'$  与  $K$  非常接近相等，所以所求正常水深为  $h_0 = 1.629\text{m}$ 。

## (2) 求临界水深

对于梯形渠道，临界水深的求解可依据临界流方程  $\frac{\alpha Q^2}{g} = \frac{A_k^3}{B_k}$ ，利用试算法求得：

先计算方程左侧值  $\frac{\alpha Q^2}{g} = \frac{20^2}{9.8} = 40.816$

则临界水深所对应的过水断面面积及水面宽度如下：

$$A_k = (b + mh_k)h_k = (3 + h_k) \times h_k$$

$$B_k = b + 2mh_k = 3 + 2h_k$$

假设水深  $h_k = 1\text{m}$

计算  $\frac{A_k^3}{B_k} = 12.8 \neq \frac{\alpha Q^2}{g} = 40.816$ , 所以临界水深  $h_k \neq 1\text{m}$

$h_k$	1	1.5	1.4	1.407	1.405
$A_k^3/B_k$	12.8	51.258	40.301	41.005	40.803

分别取  $h_k = 1\text{m}$ 、 $h_k = 1.5\text{m}$ 、 $h_k = 1.4\text{m}$ 、 $h_k = 1.407\text{m}$ , 均不能满足临界流方程。

继续假设水深  $h_k = 1.405\text{m}$

计算  $\frac{A_k^3}{B_k} = 40.803$  与  $\frac{\alpha Q^2}{g} = 40.816$  非常接近, 所以临界水深为  $h_k = 1.405\text{m}$ 。

### (3) 求临界底坡

根据所求临界水深  $h_k = 1.405\text{m}$ , 则:

过水断面面积  $A_k = (b + mh_k)h_k = (3 + 1.405) \times 1.405 = 6.189\text{m}^2$

水面宽度  $B_k = b + 2mh_k = 3 + 2 \times 1.405 = 5.81\text{m}$

湿周  $\chi_k = b + 2h_k\sqrt{1+m^2} = 3 + 2 \times 1.405 \times \sqrt{1+1^2} = 6.974\text{m}$

水力半径  $R_k = \frac{A_k}{\chi_k} = \frac{6.189}{6.974} = 0.887$

谢才系数  $C_k = \frac{1}{n} R_k^{\frac{1}{6}} = \frac{1}{0.015} \times (0.887)^{\frac{1}{6}} = 65.353$

临界底坡  $i_k = \frac{gA_k}{\alpha C_k^2 R_k B_k} = \frac{9.8 \times 6.189}{65.353^2 \times 0.887 \times 5.81} = 0.00275$

### (4) 判断水流流态

因为  $i = 0.0016 < i_k = 0.00275$ , 所以水流为缓流;

因为  $h_0 = 1.629\text{m} > h_k = 1.405\text{m}$ , 所以水流为缓流。

### 习题 7-8

解:

根据已知条件可知, 求得单宽流量:

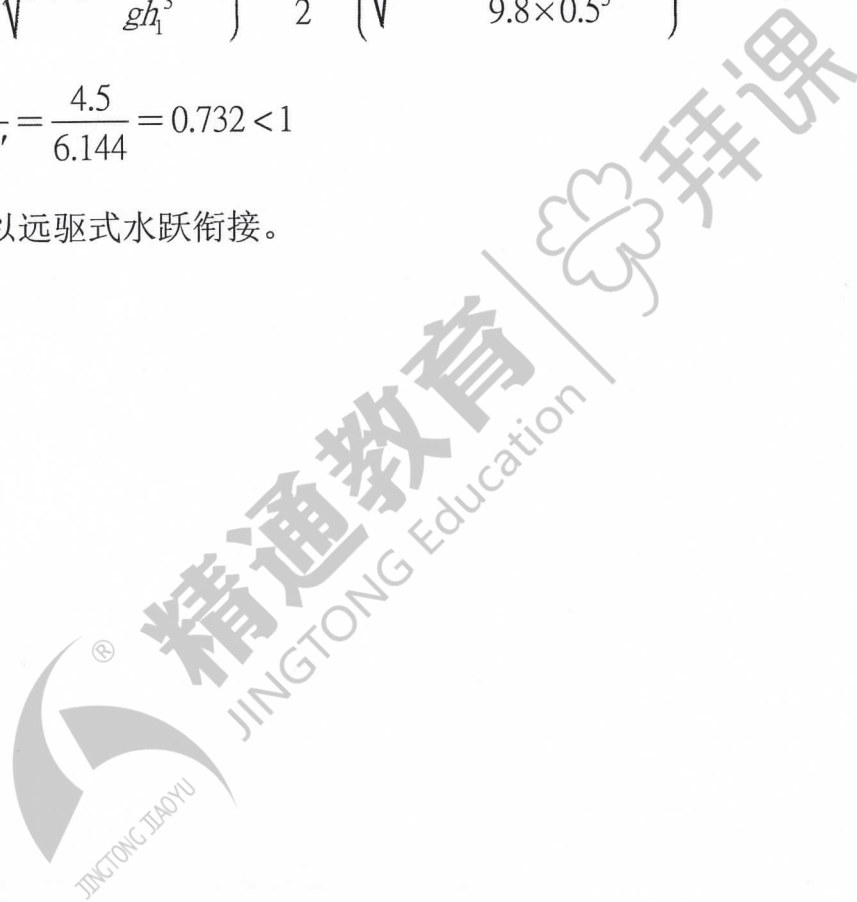
$$q = \frac{Q}{b} = \frac{50}{5} = 10 \text{m}^3/\text{s}\cdot\text{m}$$

跃后水深为:

$$h_2 = \frac{h_1}{2} \left( \sqrt{1 + 8 \times \frac{q^2}{gh_1^3}} - 1 \right) = \frac{0.5}{2} \times \left( \sqrt{1 + 8 \times \frac{10^2}{9.8 \times 0.5^3}} - 1 \right) = 6.144 \text{m}$$

$$\sigma_j = \frac{h_t}{h_c''} = \frac{4.5}{6.144} = 0.732 < 1$$

所以是以远驱式水跃衔接。



8-4.

解: 因为上游河道为近似三角形断面.

则由已知条件得, 上游水深为  $h = 267.85 - 180 = 87.85 \text{ m}$ .

$$\text{则 } A_0 = \frac{1}{2} \times 200 \times 87.85 = 8785 \text{ m}^2.$$

$$V_0 = \frac{Q_d}{A_0} = \frac{6840}{8785} = 0.779 \text{ m/s}$$

$$\text{则 } \frac{\alpha_0 V_0^2}{2g} = \frac{1 \times 0.779^2}{2 \times 9.8} = 0.03 \text{ m. 因此可忽略上游流速水头.}$$

$$\text{则 } H_0 = H_d \quad H_d - \text{堰剖面的定型设计水头.}$$

由公式 (8-2) 得

$$Q = m \epsilon_1 \epsilon_2 b \sqrt{2g} H_0^{\frac{3}{2}} \quad (8-2)$$

$$H_d = \left( \frac{Q_d}{\epsilon_1 \epsilon_2 m a n b \sqrt{2g}} \right)^{\frac{2}{3}} \quad \text{①}$$

先假设  $P_1/H_d > 1.33$ . 即查表 8-3 得  $m_a = 0.501$

假设堰为自由出流  $\epsilon_2 = 1$ .

求  $\epsilon_1$ .

查图 8-10 个身边堰形状系数  $\xi_k = 0.7$ .

查图 8-13 和表 8-5 得半圆形闸墩的形状系数  $\xi_0 = 0.45$

按式 (8-12) 计算侧收缩系数.

$$\begin{aligned} \epsilon_1 &= 1 - 0.2 [\xi_k + (n-1) \xi_0] \frac{H_0}{b} \quad (8-12) \\ &= 1 - 0.2 [0.7 + (3-1) \times 0.45] \frac{H_d}{16 \times 3} \\ &= 1 - 0.0067 H_d \quad \text{②} \end{aligned}$$

把②代入①中得

$$\begin{aligned} H_d &= \left( \frac{6840}{1 \times (1 - 0.0067 H_d) \times 0.501 \times 3 \times 16 \times \sqrt{2 \times 9.8}} \right)^{\frac{2}{3}} \\ &= \left( \frac{6425}{1 - 0.0067 H_d} \right)^{\frac{2}{3}} \end{aligned}$$

试算求得  $H_d = 17.4 \text{ m}$ .

则 堰顶高程为:  $267.85 - 17.4 = 250.45 \text{ m}$ .

上游堰高为:  $P_1 = 267.85 - 17.4 - 180 = 70.45 \text{ m}$

下游堰高为:  $P_2 = 267.85 - 17.4 - 180 = 70.45 \text{ m}$ .

复核:

$P_1/H_d = 70.45/17.4 = 4.04 > 1.33$ , 为高堰, 则  $m_d = 0.501$  是正确的。

因为下游水位  $210.5 \text{ m}$ , 小于堰顶高程  $250.45 \text{ m}$ .

所以假定自由出流也是正确的 即  $\sigma_s = 1$

$h_s < 0$ , 所以根据表 8-5 选择闸墩形状影响系数  $\xi_0$  值也是正确的,

$h_s$  — 超过堰顶的下游水深。

(2). 当上游水位高程为  $267.0 \text{ m}$  时.

则  $H = 267.0 - 250.45 = 16.55 \text{ m}$ , 忽略上游流速水头.

即  $H_0 \approx H = 16.55 \text{ m}$ .

自由出流  $\sigma_s = 1$ ,  $\xi_k = 0.7$ ,  $\xi_0 = 0.45$ .

则  $\varepsilon_1 = 1 - 0.2 [\xi_k + (n-1) \xi_0] \frac{H_0}{b}$   
 $= 1 - 0.2 [0.7 + (3-1) \times 0.45] \frac{16.55}{16 \times 3}$   
 $= 0.890$

则  $Q = m \varepsilon_1 \sigma_s b \sqrt{2g} H_0^{3/2}$   
 $= 0.501 \times 0.890 \times 1 \times 3 \times 16 \times \sqrt{2 \times 9.8} \times 16.55^{3/2}$   
 $= 6379.6 \text{ m}^3/\text{s}$

当上游水位高程为  $269.0 \text{ m}$  时.

则  $H = 269.0 - 250.45 = 18.55$ , 忽略上游流速水头.

即  $H_0 \approx H = 18.55$

自由出流  $\sigma_s = 1$ ,  $\xi_k = 0.7$ ,  $\xi_0 = 0.45$

则  $\varepsilon_1 = 1 - 0.2 [\xi_k + (n-1) \xi_0] \frac{H_0}{b} = 1 - 0.2 [0.7 + (3-1) \times 0.45] \frac{18.55}{16 \times 3}$   
 $= 0.876$

则  $Q = m \varepsilon_1 \sigma_s b \sqrt{2g} H_0^{3/2} = 0.501 \times 0.876 \times 1 \times 3 \times 16 \times \sqrt{2 \times 9.8} \times 18.55^{3/2}$   
 $= 7451.2 \text{ m}^3/\text{s}$

(2)

8-14.

解: (1) 当  $e = 1.2 \text{ m}$  时.

则有:

$$Q = \mu b e \sqrt{2gH_0} \quad (8-24)$$

因为不计行近流速水头, 则

$$H_0 = H = 45 - 39 = 6 \text{ m}$$

用公式 (8-28) 计算流量系数  $\mu$ .

$$\begin{aligned} \text{即. } \mu &= 0.60 - 0.176 \frac{e}{H} \quad (8-28) \\ &= 0.60 - 0.176 \frac{1.2}{45-39} \\ &= 0.565 \end{aligned}$$

$$\text{则. } Q = \mu b e \sqrt{2gH_0}$$

$$= 0.565 \times 5 \times 4 \times 1.2 \times \sqrt{2 \times 9.8 \times 6}$$

$$= 147.05 \text{ m}^3/\text{s}$$

(2) 当闸门全开时 即  $e = 3 \text{ m}$ .

$$\mu = 0.85$$

$$H_0 = H + \frac{\alpha_0 V_0^2}{2g} = 45 - 39 + \frac{1 \times 1^2}{2 \times 9.8} = 6.051 \text{ m}$$

$$Q = \mu b e \sqrt{2gH_0}$$

$$= 0.85 \times 5 \times 4 \times 3 \times \sqrt{2 \times 9.8 \times 6.051}$$

$$= 555.41 \text{ m}^3/\text{s}$$

